Fractional Order Control for the Grasping by Coiling

Mircea Ivanescu¹, Mihaela Florescu² and Cristian Vladu³

¹Department of Mechatronics, University of Craiova, Romania
²Department of Mechanical Engineering, University of Craiova, Romania
³Department of Electrical Engineering, University of Craiova, Romania

Abstract
The paper studies the grasping control by coiling for a class of grippers with continuum arms. First, the fractional order model of continuum arm is proposed. The viscoelastic components are assimilated by the fractional Kevin-Voigt model and the fractional-order dynamics are inferred. A hybrid control technique with two control loops for position and force control, respectively, is proposed. The Lyapunov method for position control is applied. Sector-type constraints for input torque are implemented and frequential conditions, Popov Circle Criterion, that ensure asymptotic stability, are inferred. A conventional PD controller is proposed for the force control. Numerical simulations illustrate the performances of the control system.

Keywords
Continuum arm, Fractional-order model, Control

Introduction
In this paper, a class of arms with continuum elements that performs the grasping function by coiling is analysed. This function is often met in the animal world as octopus tentacle, the elephant’s trunk or constrictor snakes. In robotic applications, this function is used to manipulate objects with geometric shapes that contain borders with concave areas (Figure 1), to objects with certain fragility and resistance to manipulate, etc. In this area of application, the architectures of continuum arm type play an important factor in determining the control laws. The specialized literature contains a large number of papers addressing the study of these models known as systems with distributed parameters. A summary of these results can be found in [1]. Among the pioneering works we can mention [2-4] which introduce the basic concepts of these models, the state equations. The dynamic models and the control laws for constrained motions are inferred in [5,6]. In [7] are discussed kinematic algorithms to avoid the obstacles of hyper-redundant manipulators. An extension of Jacobian based methods for infinite dimensional systems is presented in [8]. In [9] are studied the grasping configurations of continuum arms. Similarities of these
systems with certain classes of biological models are analysed in [10]. Kinematic control for real time implementation is discussed in [11]. Sliding mode control techniques for hyper-redundant robots are studied in [12]. Architectures based on concentric tubes for the continuum arm are presented in [13]. Spatial weight error control for a class of hyper-redundant robots is treated in [14]. Shape control of the tentacle robots is presented in [15,16].

This paper studies the grasping control by coiling for a class of grippers with continuum arms. First, the fractional order model of continuum arm is proposed. The viscoelastic components are assimilated by the fractional Kevin-Voigt model and the fractional-order dynamics are inferred. A hybrid control technique with two control loops for position and force control, respectively, is proposed. The Lyapunov method for position control is applied. Sector-type constraints for input torque are implemented and frequential conditions that ensure asymptotic stability, similar to the Popov Circle Criterion for classical systems, are inferred.

A conventional PD controller is proposed for the force control. Numerical simulations illustrate the performances of the control system.

The paper is structured as follows: Section 2 presents technological principles, section 3 treats the dynamic model, section 4 describes the control system, section 5 verifies the control techniques by numerical simulations, section 6 presents discussions and section 7 resumes the conclusions.

**Technological Principles**

The main component of the grasping system is the continuum arm that manipulates the object by exerting a sufficiently high pressure on the contact surface so that it can execute its displacement on a predetermined path. We denote this system as “Continuum Arm Grasping System (CAGS)”. CAGS can achieve any position and orientation in 3-D space to surround the manipulated load on the surface of different geometric shapes. CAGS constitutes the terminal component of a conventional robot intended for current part handling functions.

**Figure 1:** Grasping by coiling.

**Figure 2:** Arm segment technological structure.
The present work is focused only on the control of the grasping operations.

The main component of the CAGS mechanical structure is a long flexible backbone. It is surrounded by an elastic envelope and a viscoelastic solution fill the whole system ensuring a large flexibility (Figure 2). The actuation system consists of antagonistic Bowden cables with DC motor driving. The mechanical architecture is composed of several segments with independent Bowden cable driving system that allows a decoupled control strategy for each arm segment. The measuring system consists of two components: A measuring system of the geometric parameters of the orientation of the arm and one for measuring the compression forces during the grasping operation. The first system is achieved by measuring the lengths of the Bowden cables [16]. The second system is consisted by an array of Force Sensitive Resistors (FSR) (Figure 2). They are constructed of several thin flexible layers that varies in resistance as pressure is applied and released [17].

The backbone of the arm is defined by a 3-dimensional curve $C$ and a length parameter $s$, $s \in \Omega$, $\Omega = [0, l]$, where $l$ is the length of the arm. The arm orientation is given by two angles $u_1$, $u_2 \in \Gamma(\Omega) \subset L_2(\Omega)$, defined by a right-handed orthonormal basis vector $\{i_x, i_y, i_z\}$ is parallel to a base coordinate frame, Figure 3.

**Dynamic Model**

The dynamic behaviour of the CAGS is determined by its structure with distributed mass and by the viscoelastic component characterized by the Kevin-Voigt fractional order model defined by the stress-strain $(f, \varphi)$ relation

$$f(t) = c_e \varphi(t) + c_v \left(D_0^\beta \varphi(t)\right)$$  \hspace{1cm} (1)

Where $(D_0^\beta \varphi(t))$ is the Caputo fractional derivative of order $\beta$. The dynamic model is derived by using Lagrange equations [5,14] for this class of hyper-redundant arms. We consider a light weight arm with following parameters: $I_\rho$ is inertial rotational density, $E$ is equivalent elastic modulus and $I$ is the moment of inertia. It is assumed that during the grasping operation, the arm is subjected to a disturbance, the moment $m$, distributed along the contact surface. In terms of the orientation variables $u_1, u_2$ this model can be rewritten as

$$I_\rho \ddot{u} = E I_{ss} u_{ss} - C_v D^\beta u - C_m u + h$$ \hspace{1cm} (2)

Where

$$u = u(t, s), \quad u_u = \frac{\partial^2 u(t, s)}{\partial s^2}, \quad u = [u_1 u_2]^T, \quad u_1, u_2 \in L_2(\Omega),$$

$$D^\beta u = i D_0^\beta u(t, s), \quad I_\rho = \text{diag}(I_{\rho_1}, I_{\rho_2}),$$

$$EI = \text{diag}(EI_1, EI_2), \quad C_v = \text{diag}(c_{v_1}, c_{v_2}),$$

$$C_m = \text{diag}(c_{m_1}, c_{m_2}), \quad D^\beta u$$

is the shortened notation for the Caputo fractional derivative. The initial conditions are

$$u(0, s) = u_0(s)$$

The boundary conditions are determined by the

![Figure 3: Geometrical parameters of the arm segment.](image-url)
driving moments,
\[ u(t,0) = 0; u_i(t,l) = \tau(t,l) \quad (3) \]

Where, \( \tau = [\tau_1, \tau_2]^T \). Technical constraints impose the sector constraints on the driving torques,
\[ \alpha u \leq \tau \leq \beta u \quad (4) \]

The nonlinear term \( h(u) \) is determined by the gravitational component and satisfies the constraints \([14,15]\).
\[ \|uh(u)\| \leq \chi u^T u \quad (5) \]

Where, \( \chi \) is a positive constant. The system output is evaluated as a weighted average
\[ y(t) = \frac{1}{0} \psi(s)u(t,s)ds \quad (6) \]

Where \( y = [y_1, y_2]^T \) and \( \psi(s) \) is an eigenfunction of the operator \( \frac{\partial^2}{\partial s^2} \). The aim of the control problem is to ensure the movement to the target position represented by the curve \( C_t \),
\[ u_i(s) = \{u(s) : s \in C_i\} \quad (7) \]

The evolution error will be
\[ \mathcal{E}(t) = \frac{1}{0} \psi(s)(u_i(s) - u(t,s))ds \quad (8) \]

The dynamic model (2)-(3) becomes
\[ f_i(t) = -C_D \delta e_i(t) - \alpha e_i(t) + EI e_i(t) - h^*, i = 1,2 \quad (9) \]

Where
\[ a = \left(\frac{\pi^2}{4l^2}EI + C_e\right) \quad (10) \]

\[ e_i(0) = e_i' \quad (11) \]

The fractional order model will be \([18-21]\) (for \( \beta = 0.5 \)).
\[ e_i(t) = e_i', D^\beta e_i(t) = e_i'(t), D^\beta e_i(t) = e_i''(t), ..., i = 1,2 \quad (12) \]

The following fractional order model can be inferred from (9)-(12) (for simplification of notations the index \( i \) is eliminated starting from here),
\[ D^\beta e(t) = Ae(t) + b\tau(t) + dm(t) + h^* \quad (13) \]
\[ z(t) = c^T e(t) \quad (14) \]

Where
\[ e(t) = [e_1(t) e_2(t) e_3(t) e_4(t)]^T \quad (15) \]
\[ A = \begin{bmatrix} 0 & 1 & 0 & 0 \\ 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 1 \\ -\frac{1}{I_c} & -\frac{1}{I_c} & 0 & 0 \end{bmatrix} \quad (16) \]
\[ c = [1 0 0 0]^T; b = \begin{bmatrix} 0 & 0 & \frac{1}{I_c} \end{bmatrix}^T \quad (17) \]

\[ h^* = \begin{bmatrix} 0 & 0 & 0 & 1 \end{bmatrix} \quad (18) \]

**Grasping Control System**

The control of the grasping operation is a hybrid control system, position-force control that comprises two phases. In the first phase, the free movement of the arm, without contact with the object,
is controlled to the target position $u_t$ (Figure 4), in the second phase; the grasping operation is performed, the arm exerting on the object the needed compression force. The control system is shown in Figure 5.

Two cases will be analyzed: The control system for a free movement to a desired position and the control system for the control of the grasping compression forces.

**Grasping position control**

Dynamic model is rewritten in terms of error (13)-(18) as

$$D^2 e(t) = A e(t) + b \tau(t) + h^* (e)$$  \hspace{1cm} (19)

Consider the control law

$$\tau(t) = -k e(t)$$  \hspace{1cm} (20)

The force constraints (4) determine sector type restrictions on the gain $k$, $k \in [k_\alpha, k_\beta]$  \hspace{1cm} (21)

Where $k_\alpha, k_\beta$ define the new sector parameters.

**Algorithm 1.** The dynamic system (19), with the control variable (20), (21) is asymptotically stable if:

a) $A^* = (A - R)$ is Hurwitz stable, where $R = R^T > 0$  \hspace{1cm} (22)

b) $\text{Re}(G(j\omega)) \geq -\sqrt{\sigma}$, $\forall \omega \in (-\infty, +\infty)$  \hspace{1cm} (23)

c) $\alpha^* = -2\lambda_{\max}(PR) > 0$  \hspace{1cm} (24)

Where the following notations were used: $G(j\omega) = C^T (j\omega I - (A^* + \sigma I))^{-1} B$, $Q = qq^T$, $P$ is a solution of Lyapunov equation, $I$ is the unit matrix, $\sigma = \sqrt{(q + k\sqrt{\sigma c})(q + k\sqrt{\sigma c})^T}$ and $\sigma$ is a constant that satisfies the condition $k \sigma \leq 1$  \hspace{1cm} (25)

Where, from (21) yields

$$\sigma \in [\sigma_\alpha, \sigma_\beta]$$  \hspace{1cm} (26)

The frequency inequality (23) can be interpreted in terms of the constraints (21), (26) as [22],

![Figure 5: Hybrid control system.](image)

![Figure 6: Circle criterion for light weight continuum arm.](image)
\[ Re \left[ \frac{\sigma_{\beta} + G(j\omega)}{\alpha + G(j\omega)} \right] > 0, \forall \omega \in (-\infty, +\infty) \]  

(27)

Consider that \( C(\sigma_{\alpha}, \sigma_{\beta}) \) to be the circle in the complex plane (Figure 6) defined by the points \(-\sigma_{\alpha} + j\omega\) and \(-\sigma_{\beta} + j\omega\). In this case, the condition (26) can be defined as: all points of plot of \( G(j\omega) \) must be outside the circle \( C(\sigma_{\alpha}, \sigma_{\beta}) \), or as the plot of \( G(j\omega) \) does not intersect \( C(\sigma_{\alpha}, \sigma_{\beta}) \) (Figure 6).

Because \( G(j\omega) \) defines the frequency characteristics of the arm and \( C(\sigma_{\alpha}, \sigma_{\beta}) \) represents the input constraints, the stability condition can be rewritten as: "the both domains must not intersect".

\[ G \cap C(\sigma_{\alpha}, \sigma_{\beta}) = \emptyset \]  

(28)

The relation (28) can be considered as an extension of the Popov's Circle Criterion for this class of fractional order models [22].

**Convergence Algorithm Proof.** The following Lyapunov function is used

\[ V(e) = e^T Pe \]  

(29)

Where, \( P = P^T > 0 \). This Lyapunov function verifies asymptotic stability conditions [19,23,24]. From [21-26], the fractional derivative of (29) becomes

\[ D^{\beta} V(e) \leq \left( D^{\beta} e^T \right) Pe + e^T P \left( D^{\beta} e \right) \]  

(30)

By evaluating this inequality along the solutions of (19), results

\[ D^{\beta} V(e) \leq e^T \left( A^T P + PA \right) e + 2e^T Pb \tau \]  

(31)

Considering (22), the relation (31) becomes

\[ D^{\beta} V(e) \leq e^T \left( (A - R)^T P + P(A - R) \right) e + 2e^T Pb \tau + 2e^T PRe \]  

(32)

From (19), (21), (25), this inequality is rewritten as

\[ D^{\beta} V(e) \leq e^T \left( (A - R)^T P + P(A - R) \right) e + 2e^T (Pb \cdot \frac{1}{2} \tau + \sigma_{\alpha} + 2e^T PRe \]  

(33)

Applying Yakubovici-Kalman-Popov (YKP) Lemma [22], yields

\[ e^T \left( (A - R)^T P + P(A - R) \right) e = -qq^T \]  

(34)

\[ Pb - \frac{1}{2} c = \sqrt{\sigma_{\alpha} q} \]  

(35)

Substituting this result into (33), after simple calculations, considering the control law (19), it follows that

\[ D^{\beta} V(e) \leq -e^T (q + k\sqrt{\sigma_{\alpha} q}) (q + k\sqrt{\sigma_{\alpha} q}) e + \rho_{\alpha} ee \]  

(36)

or, by (24) [25,26],

\[ D^{\beta} V(e) \leq -\alpha e^T e \]  

QED  

(37)

**Grasping force control**

Consider that the stress-strain \(( f, \varphi) \) relation of the Kelvin-Voigt model (1) can be approximated by \(( f, \varphi) \approx (f, \Delta u) \) (38)

Assume that \( f(t, s) \) is the measured distributed force, \( f = \left[ f_1, f_2 \right] \), \( f_d(s) \) is the desired, needed grasping force. The weighted average of the error is evaluated from (6) as

\[ e_f(t) = \int_0^l \psi(s) (f_d(s) - f(t, s)) \, ds \]  

(39)

The force dynamics can be approximated from (1), (38), (40) using the same procedure as developed in (8),

\[ \dot{\varphi}_i = \alpha_{ij} \varphi_i + \beta_{ij} \tau_i + \delta_i, i = 1, 2 \]  

(40)

\[ e_f = \left[ e_{f1}, e_{f2} \right]^T \]  

(41)

Where \( \alpha_{ij}, \beta_{ij} \) are the equivalent coefficients obtained from (8).

**Algorithm 2.** The force control system is asymptotically stable with the following PD control law

\[ \tau_i(t) = -k_{f1} e_{f1}(t) - k_{f2} e_{f2}, i = 1, 2 \]  

(42)

Where \( k_{f1}, k_{f2} \) are positive gains of the controller that satisfy the conditions:

\[ a_{ij} p_1 + k_{f1} b_{ij} p_2 - \chi > 0 \]  

(43)

\[ k_{f2} b_{p2} - p_1 > 0, i = 1, 2 \]  

(44)

Where \( p_1, p_2, p_3 \) are positive constants \( p_1 > p_2, p_2 > p_3 \), that verify

\[ p_1 - a_{ij} p_2 - k_{f1} b_{ij} p_2 - k_{f2} b_{ij} p_3 + p_2 \chi = 0 \]  

(45)

Proof: The proof is simple and is obtained by Lyapunov conventional techniques [22,27].

**Numerical Simulations**

Consider a grasping control problem for a 2-D motion in a horizontal plane (Figure 7). The following parameters characterize the mechanical system: the length of the arm segment \( l = 0.15 \, \text{m} \), the rotational inertial density \( I_\rho = 0.032 \, \text{kg} \cdot \text{m}^2 \), the bending stiffness \( EI = 1.2 \, \text{N} \cdot \text{m}^{-1} \), equivalent Kelvin-Voigt model coefficients: \( c_v = 0.26 \, \text{Nm} / \text{rad}, \ c_e = 2.1 \, \text{Nm} \cdot \text{rad}^{-1} \). The fractional order model (12)-(15) will be
MATLAB/SIMULINK and techniques based on the Mittag-Leffler functions are used for the simulation \([31,32]\). A weighting eigenfunction 

\[
\psi(s) = \sin\left(\frac{\pi x}{2T}\right)
\]

is used. Initial position is 

\[
\left(\begin{array}{c}
0 \\
0 \\
0.15
\end{array}\right)
\]

and the target position is defined as 

\[
\left(\begin{array}{c}
6 \\
\pi
\end{array}\right)
\]

In Figure 8 are shown the error trajectories for position 

\[
e_1(t) = e(t)
\]

and velocity 

\[
e_3(t) = \dot{e}(t)
\]

The good quality of motion can be remarked from this imagine.

The grasping force control is obtained for 

\[
f_j(s) = 150 \text{ Nm}, s \in [0,0.15]\]

with a spatial weighting average 

\[
f_j \sim 24 \text{ Nm}.
\]

A control law (42) with 

\[
k_{f1} = 18, k_{f2} = 0.7
\]

is used. The trajectory of the

\[
A = \begin{bmatrix}
0 & 1 & 0 & 0 \\
0 & 0 & 1 & 0 \\
0 & 0 & 0 & 1 \\
-249 & -56.8 & 0 & 0
\end{bmatrix}
\]

\[
b = \begin{bmatrix}
0 & 0 & 0 & 130
\end{bmatrix}^T, c = \begin{bmatrix}
1 & 0 & 0 & 0
\end{bmatrix}^T
\]

From (6), the constraints of the compression forces 

\[
m(s) = 100 \text{ Nm}, \beta_m = 5000 \text{ Nm}
\]

The system is fractional order stable but is not Hurwitz stable [28-30]. The matrix 

\[
R = \text{diag}(4.5,4.5,4.5,4.5)
\]

is selected to satisfy the condition (22) and the behaviour of the non-linear model is characterized by 

\[
\chi = 2.
\]

A control law (20) is applied with 

\[
k' = 4, k_u = 2, k_\beta = 5, k_\rho = 5, \sigma_\rho = 0.2.
\]

The condition (24) is verified for 

\[
\lambda_{\text{max}(Q)} = 14.5, \lambda_{\text{max}(P)} = 0.725, \|P R\| = 2.47, \rho_\beta = 2.7.
\]

Citation: Ivanescu M, Florecu M, Vladu C (2020) Fractional Order Control for the Grasping by Coiling. Int J Robot Eng 5:024
force error $e_f(t)$ (39) is shown in Figure 9.

**Discussion**

The dynamic model of the continuum arm is associated with the lightweight model, for which the gravitational component has a small influence. If this component increases, the stability conditions are getting worse.

In Figure 10 are shown two plots of the linear and nonlinear continuum arm models, respectively. We remark that the nonlinear component moves the plot of $\tilde{G}(j\omega)$ to the left, toward the circle $C$ which leads to a decrease in stability. For high value of nonlinearity coefficient $\chi$, when $\tilde{G}(j\omega)$ intersects the circle $C$, the system becomes unstable.

**Conclusions**

The paper studies the grasping control by coiling for a class of grippers with continuum arms. First, the fractional order model of continuum arm is proposed. The viscoelastic components are assimilated by the fractional Kevin-Voigt model and the fractional-order dynamics are inferred. A hybrid control technique with two control loops for position and force control, respectively, is proposed. The Lyapunov method for position control is applied. Sector-type constraints for input torque are implemented and frequent conditions that ensure asymptotic stability, similar to the Popov Circle Criterion for classical systems, are inferred. A conventional PD controller is proposed for the force control. Numerical simulations illustrate the performances of the control system.

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